

# **SIMULATION AND OPTIMIZATION OF MEMS PIEZOELECTRIC ENERGY HARVESTER WITH A NON-TRADITIONAL GEOMETRY**

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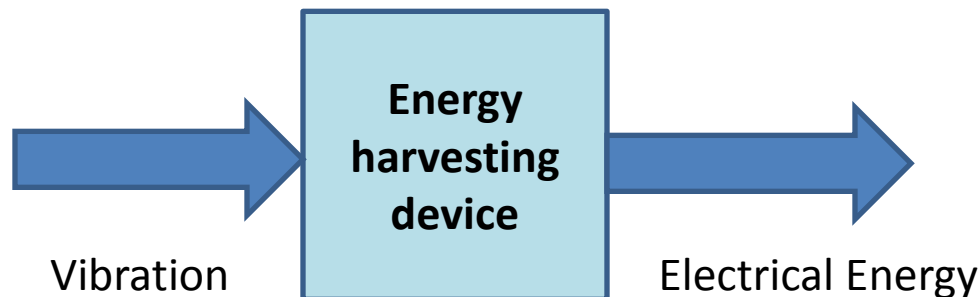
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# Introduction

- The energy harvesting devices convert ambient energy into electrical energy .
- It is the concept by which energy is captured, stored and utilised.
- Ambient energy is available in the form of vibration, light, temperature gradient etc.
- Among these energy, mechanical vibration is the most widespread and wasted energy in the environment.



# Theory

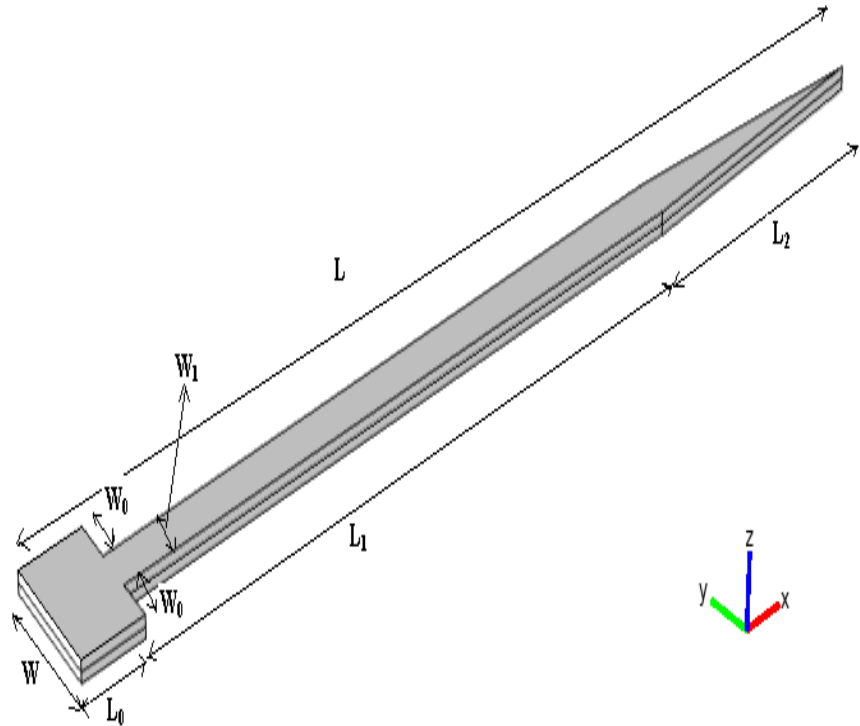
- Conversion of Mechanical vibration into electrical energy
  - Electromagnetic
  - Electrostatic
  - Piezoelectric.
- Piezoelectric generators are mostly used because piezoelectric materials have the advantage of large power and ease of application.
- Direct piezoelectric effect: surface charge induced by a mechanical stress.
- The most studied energy harvesters are based on the piezoelectric effect and are made with MEMS technology.
- The geometry of piezoelectric cantilever beam greatly affects its vibration energy harvesting ability .
- In this paper MEMS based energy harvester with a non-traditional geometry is designed and simulated with COMSOL for the conversion of mechanical into electrical energy.
- Also the results are compared with other geometries such as rectangular and triangular.

# Use of COMSOL

## Geometry

The geometry consists of two subdomains,

1. substrate layer- stainless steel.
2. piezoelectric layer- active layer of unimorph .

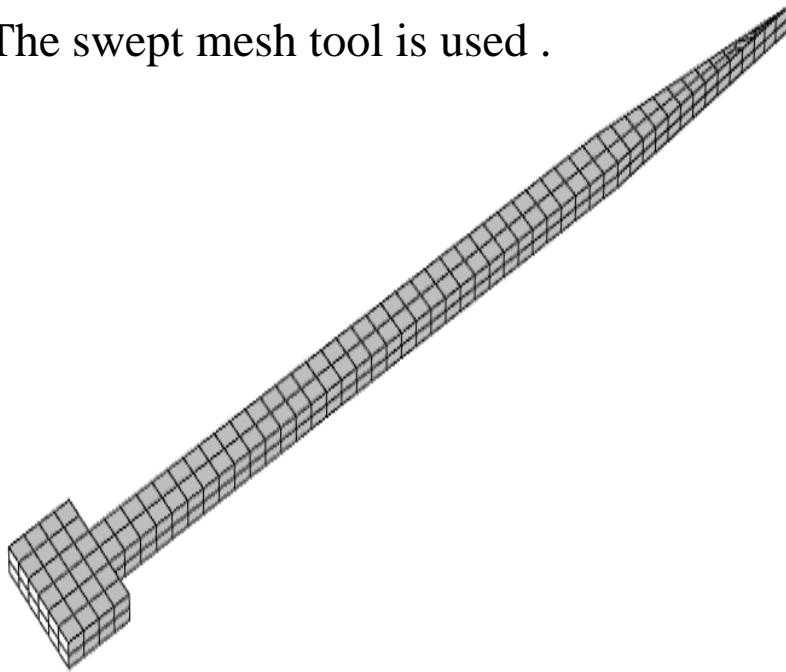


**Figure 1.** Structure of piezoelectric energy harvester with non-traditional geometry.  $L=27000\mu\text{m}$ ,  $L_0=2000\mu\text{m}$ ,  $L_1=18000\mu\text{m}$ ,  $L_2=7000\mu\text{m}$ ,  $W=3000\mu\text{m}$ ,  $W_0=W_1=1000\mu\text{m}$ ,  $T_0=200\mu\text{m}$ ,  $T_1=210\mu\text{m}$ .

# Meshing and Governing Equations

## Meshing

- The mesh consists of 238 quad elements for a total number of degrees of freedom 10639.
- The swept mesh tool is used .



**Figure 2.** Piezoelectric Energy Harvester mesh.

## Piezoelectric Equations

$$S = s^E T + d E \quad (1)$$

$$D = \varepsilon^T E + d T$$

Where

S - the mechanical strain vector

$s^E$  -elastic compliance tensor ( $\text{Pa}^{-1}$ )

T- mechanical stress vector ( $\text{Nm}^{-2}$ )

D -elastic displacement vector ( $\text{Cm}^{-2}$ )

$\varepsilon^T$  -the dielectric permittivity tensor ( $\text{Fm}^{-1}$ )

E- the electric field vector ( $\text{Vm}^{-1}$ )

d- the transverse piezoelectric coefficient tensor ( $\text{CN}^{-1}$ ).

For the **substrate layer** only mechanical behaviour is considered using stress-strain relationship.

$$S = sT \quad (2)$$

s is the compliance of stainless steel substrate.

# Subdomain settings

- The material parameters of the substrate are as follows: its density  $\rho=7850$  kg/m<sup>3</sup>, Young's modulus  $E=200 \times 10^9$  Pa, Poisson's ratio  $\mu=0.33$ .
- The active layer of unimorph is modelled using the following set of properties.

-Elastic compliance tensor

$$\mathbf{S}^E = \begin{bmatrix} 50 & -20 & -20 & 0 & 0 & 0 \\ -20 & 50 & -20 & 0 & 0 & 0 \\ -20 & -20 & 50 & 0 & 0 & 0 \\ 0 & 0 & 0 & 70 & 0 & 0 \\ 0 & 0 & 0 & 0 & 70 & 0 \\ 0 & 0 & 0 & 0 & 0 & 70 \end{bmatrix} \times 10^{-12} \text{ Pa}^{-1}$$

-Piezoelectric tensor

$$\mathbf{d} = \begin{bmatrix} 0 & 0 & 0 & 0 & 11 & 0 \\ 0 & 0 & 0 & 11 & 0 & 0 \\ -2.5 & -2.5 & 5 & 0 & 0 & 0 \end{bmatrix} \times 10^{-12} \text{ CN}^{-1}$$

-Relative permittivity matrix

$$\boldsymbol{\varepsilon}^T = \begin{bmatrix} 50 & 0 & 0 \\ 0 & 50 & 0 \\ 0 & 0 & 50 \end{bmatrix} \times \boldsymbol{\varepsilon}^0$$

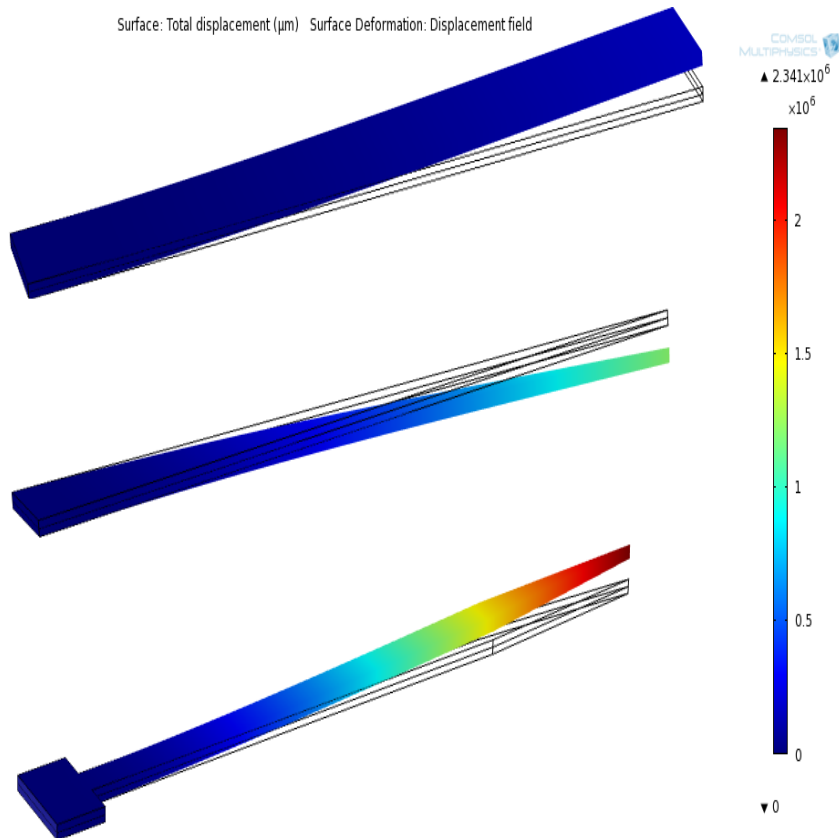
-Density  $\rho = 3000 \text{ kg m}^{-3}$

# Boundary conditions

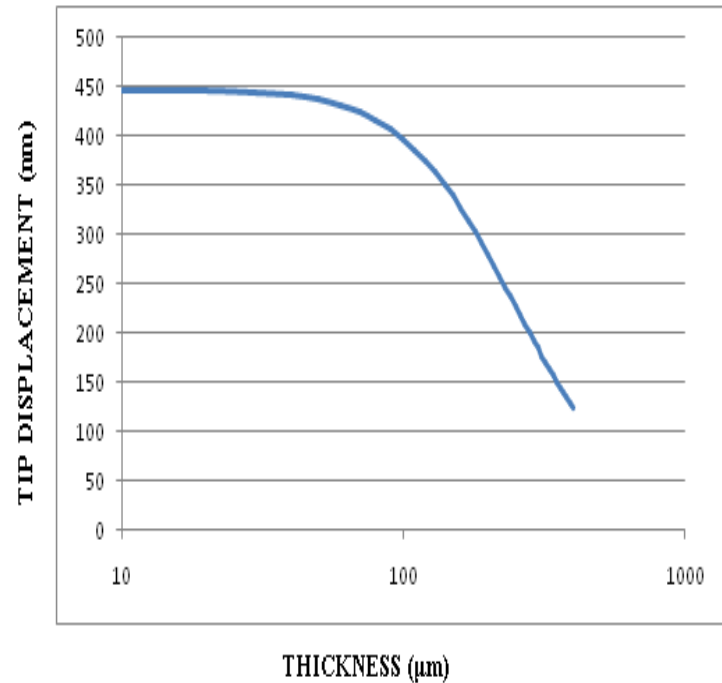
- **Vertical acceleration :**
  - body load  $F_Z = \rho a$  in each subdomain,  $a = 0.1g$  and  $\rho$  is the density of the material .
- One end of the unimorph cantilever is fixed while other is free for vibration.
  - fixed constraint condition is applied for the vertical faces of both the layers.
  - while all other faces are free of displacement.
- **Electrostatic boundary conditions :**
  - upper and lower face of PZT layer are selected as floating and ground potentials respectively .
  - while all other faces of piezoelectric layer are kept as zero charge.
- **Mesh boundary conditions:**
  - to optimize the thickness of the PZT layer, Moving mesh application mode is used.
  - bottom face: clamped,
  - vertical faces: clamped along thickness,
    - upper surface: tangentially constrained and displaced in the normal direction to the surface by a given displacement ( $\Delta$ thickness).
  - deltaThickness* is changed from  $10\mu\text{m}$  to  $400\mu\text{m}$  obtaining parameterized moving mesh.



# Modelling and Optimization

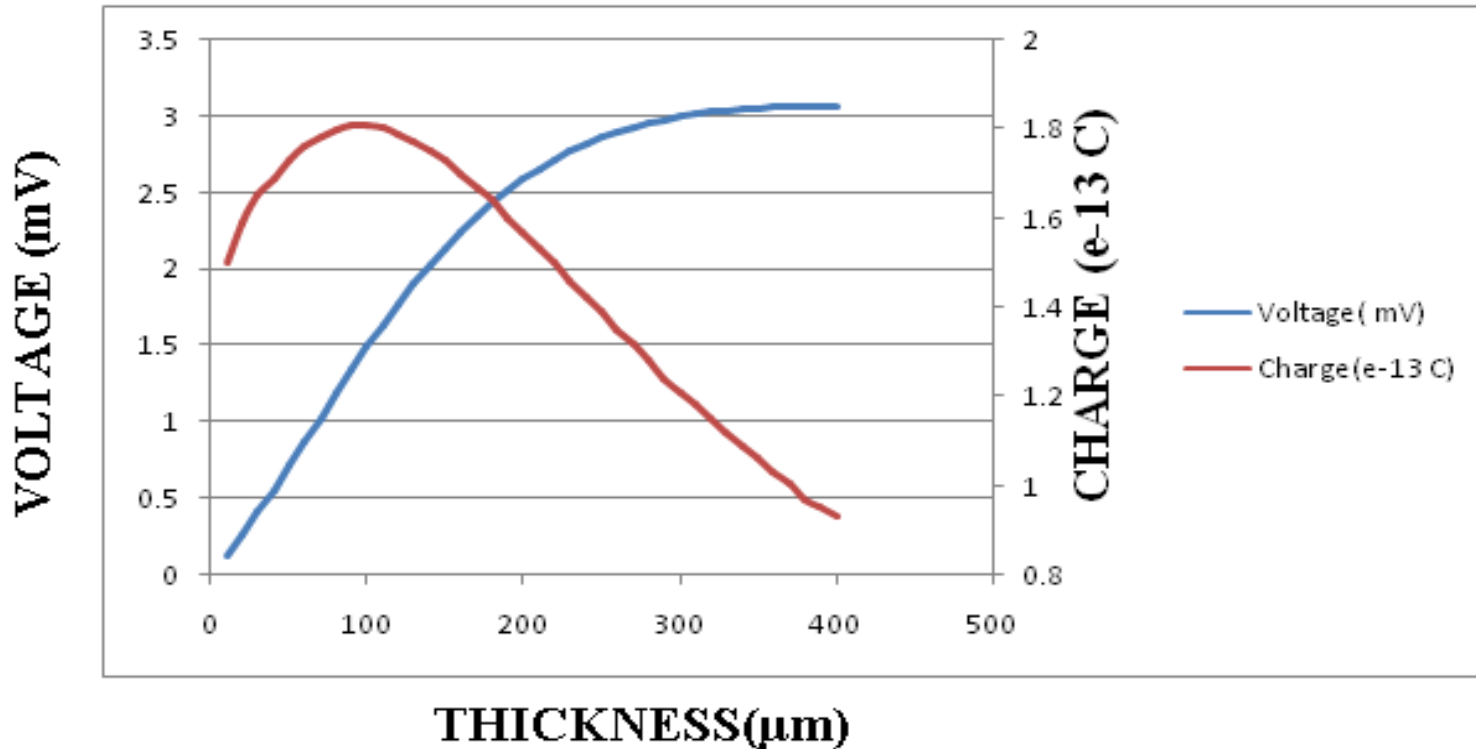


**Figure 3** FEM modelling of rectangular, triangular and non-traditional geometries.



**Figure 4** Tip displacement (nm) Vs Thickness( $\mu\text{m}$ ).

# Results-output voltage

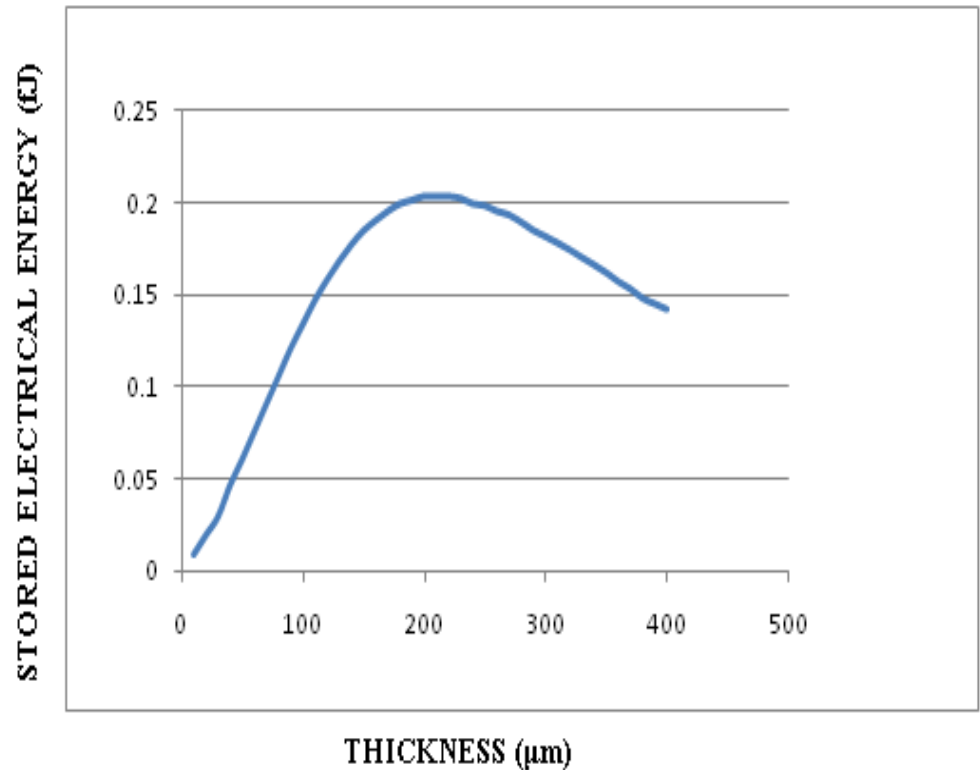


**Figure 5** voltage (mV) Vs thickness ( $\mu\text{m}$ ) and charge (e-13) Vs thickness ( $\mu\text{m}$ ).

# Results-Stored energy

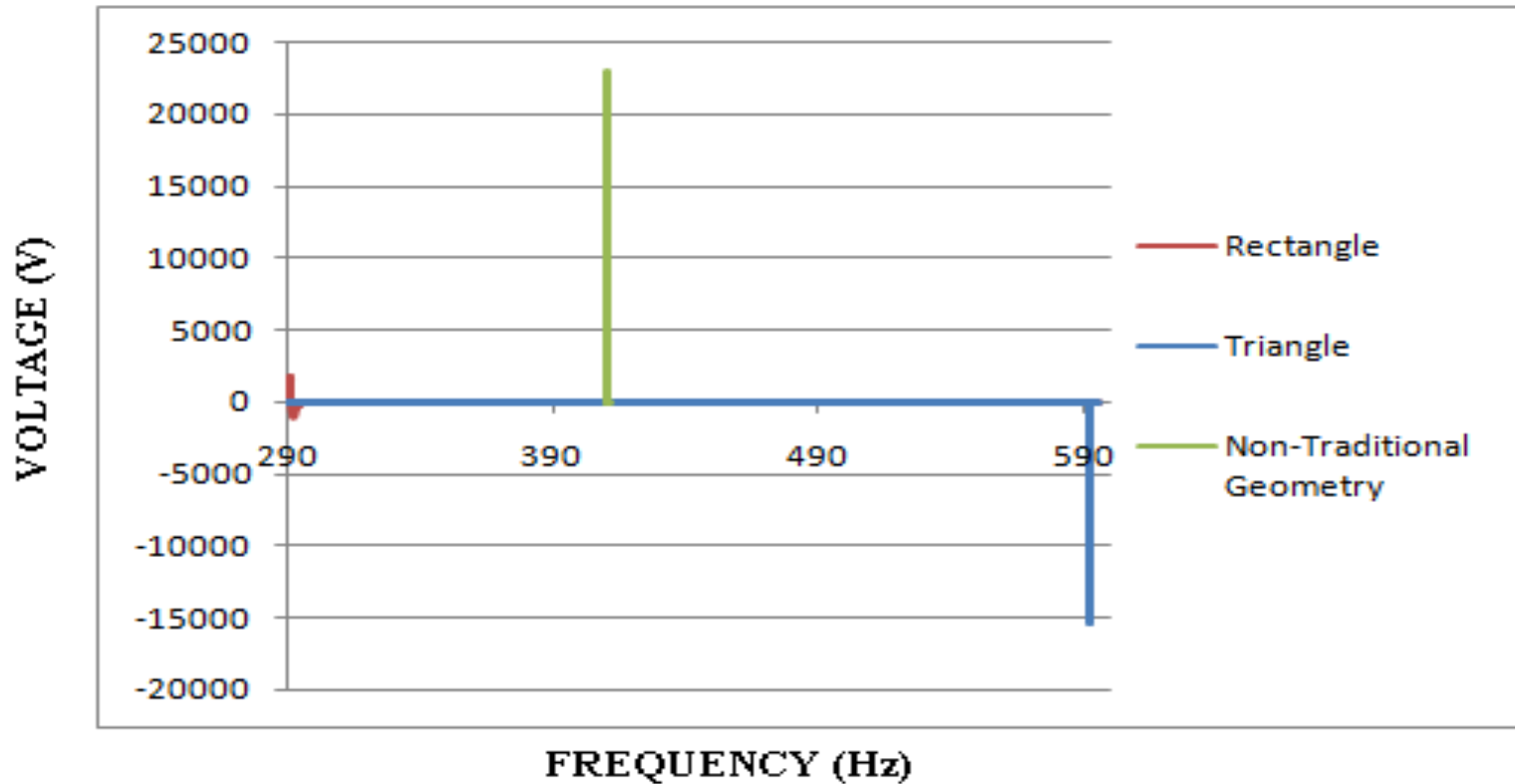
Stored electrical energy

$$E = \frac{1}{2} QV$$



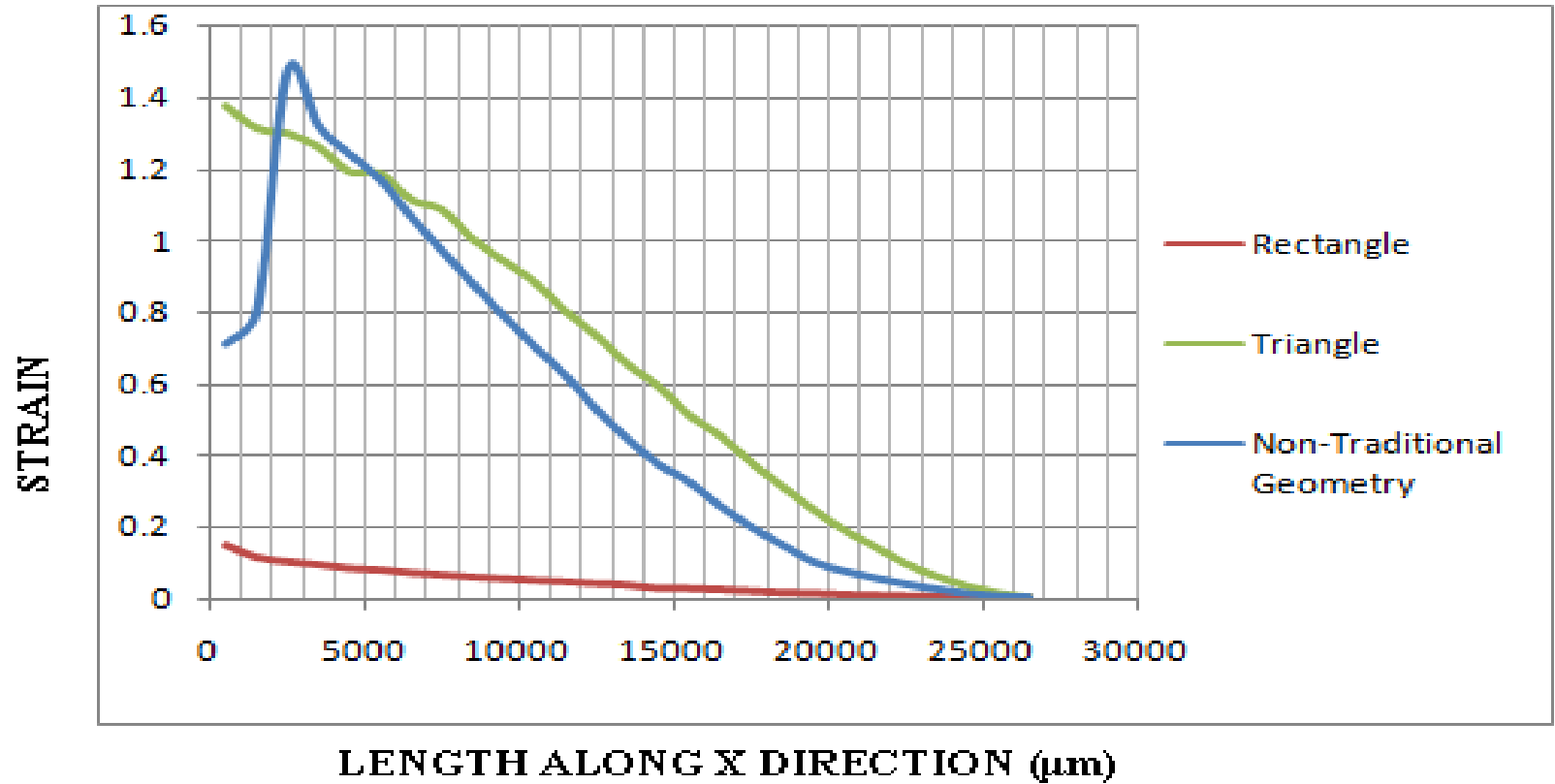
**Figure 6** Stored electrical energy (fJ) Vs thickness (μm).

# Results-Frequency analysis



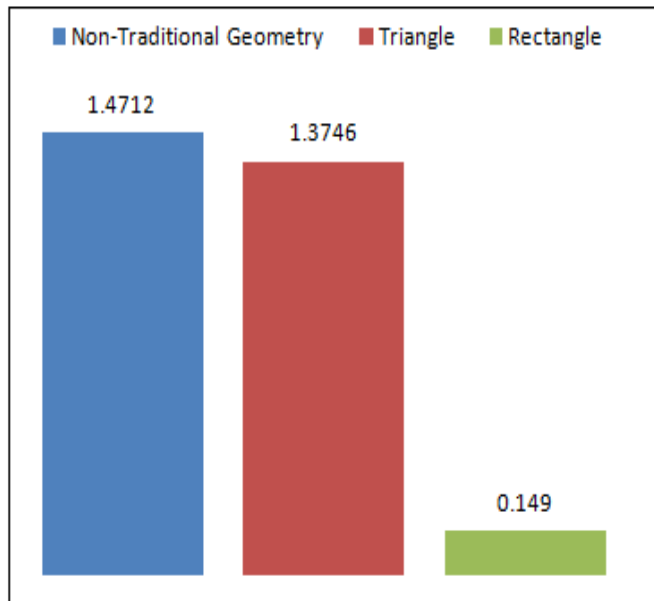
**Figure 7** Voltage (V) Vs Frequency (Hz) for three different geometries.

# Results- Strain analysis

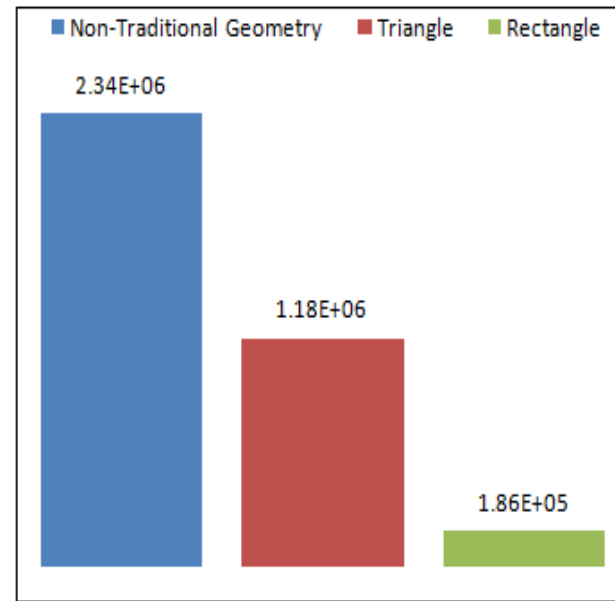


**Figure 8** Strain curves of three different geometries along X direction.

# Performance comparison



a) Strain.



b) Deformation.

# Summary

- A piezoelectric energy harvester with non-traditional geometry was designed and simulated in COMSOL Multiphysics.
- The thickness of PZT layer was optimized to give maximum stored electrical energy.
- Frequency analysis and strain analysis were carried out for the optimized thickness of  $210\mu\text{m}$ .
- Simulation results demonstrate that the piezoelectric energy harvester with non-traditional geometry improves strain and generate more voltage at resonant frequency than the rectangular and triangular piezoelectric energy harvester.
- The simulation results suggest that such structures can be used for energy generation in wireless sensor networks.

# References

- [1] Z.S.Chen, Y.M.Yang and G.Q.Deng, “Analytical and Experimental Study on Vibration Energy Harvesting Behaviors of Piezoelectric Cantilevers with Different Geometries” International conference on sustainable power generation and supply, 1 - 6 ( 2009).
- [2] M.Guizzetti, V.Ferrari, D.Marioli and T.Zawada, “Thickness optimization of a piezoelectric converter for energy harvesting,” *Proceedings of the COMSOL Conference*, 2009.
- [3] Suyog N Jagtap and Roy Paily, “Geometry Optimization of a MEMS-based Energy Harvesting Device” Proceeding of the 2011 IEEE Students' Technology Symposium, 265 - 269 (2011), IIT Kharagpur.



Thank You !

Questions???