

# Time Dependent Sliding Performance of a Hyperelastic Seal

Presentation by

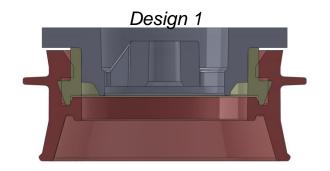
Drs. Helger van Halewijn

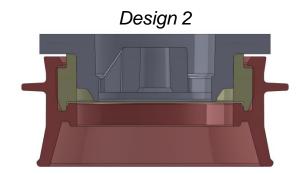


#### Introduction

- For an automatic coffee machine a part inside is used for regular maitenance.
- The part consists of an hyper-elastic material and is attached to a metal ring.
- Use has been made of COMSOL, with mechanical module, hyper-elastic material is included in the model.
- Operator should apply a force between 10 14 N.

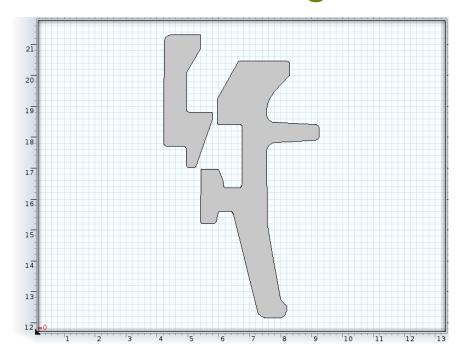
## **Designs**

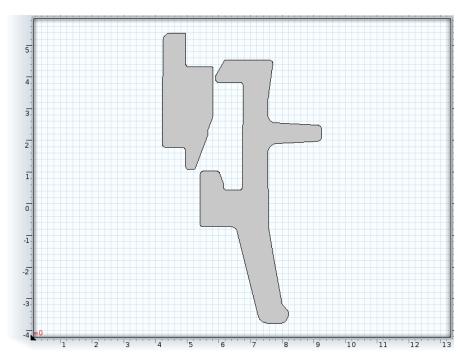




- Silicone ring can be pulled off the metal ring.
- Two designs should be simulated on force development during time.
- Force should be in the order of 10-14 N.
- Friction coefficient has been tested: μ = 2.

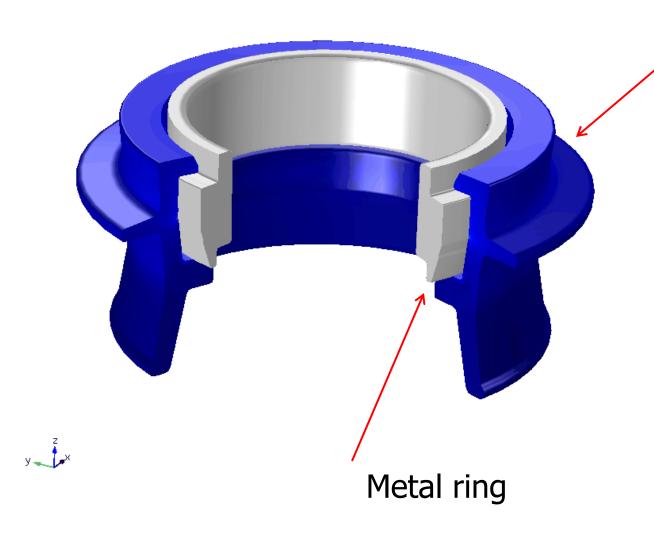
## Design1 & 2







## Overview of design 2



Hyper-elastic seal

- Question:
- Simulate the firction force during dismantling of system.
- Complications

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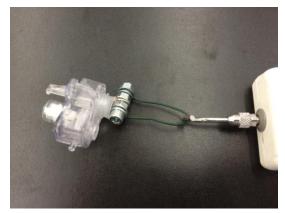
- 1. Stick slip
- 2. Pre-stressed



## Laboratory tests on friction

 Slide pack on table: Should withstand sliding a pack of 5 kg on high friction surface (specify)





b) Tests reveal a friction coefficient of 2.

### COMSOL Solid mechanics module

- Use has been made of
  - Contact mechanics with friction.
  - Hyper-elastic node: Neo-Hookean.
- Study 1:
  - Assembly of system, in order to include the prestress.
- Study 2:
  - Move the elastic seal over the metal part, with friction.
  - Time dependant study.

## **Material data**

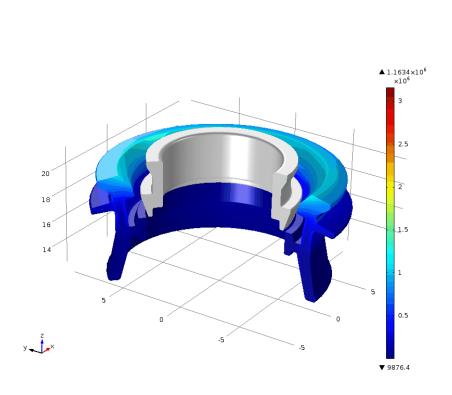
#### DATA TABLE FOR: Polymers: Elastomers: Silicone Rubber

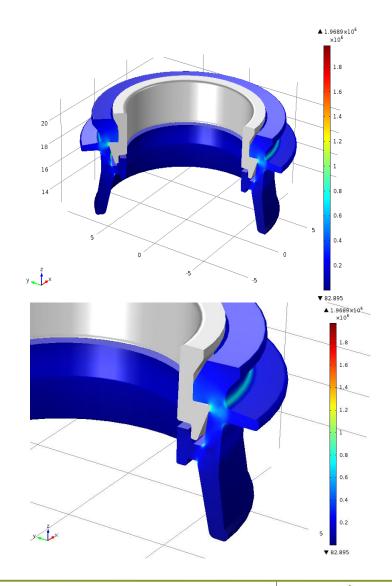
Mechani	cal Properties			
	Quantity	Value	Unit	
	Young's modulus	1 - 5	MPa	
	Tensile strength	5 - 8	MPa	
	Elongation	200 - 800	%	
Physical	Properties			
	Quantity	Value	Unit	
	Thermal expansion	8.1 - 8.1	e-6/K	
	Thermal conductivity	0.22 - 0.22	W/m.K	
	Glass temperature	-120123	°C	
	Service temperature	-70 - 250	°C	
	Density	1250 - 1250	kg/m <sup>3</sup>	
	Resistivity	1e+19 - 1e+21	Ohm.mm <sup>2</sup> /m	
	Dielectric loss factor	0.001 - 0.01		
General				
	Remarks Silicones are not or difficult to bor		d with an adhesive. Bonding is only possible with cyan	10-acrylaat af
		the application of a primer. With cy	ano-acrylaat it is not possible to bond with glass, stone	e and concre



**SAMPLE: ELASTOMERS** 

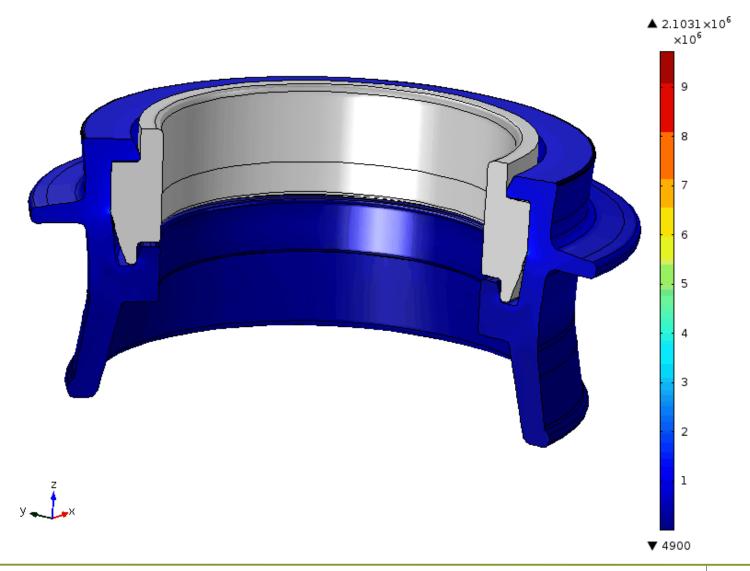
### Preparation of simulation. Assembling Design 1



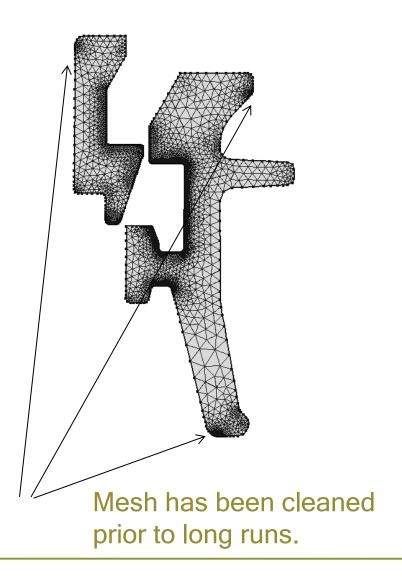


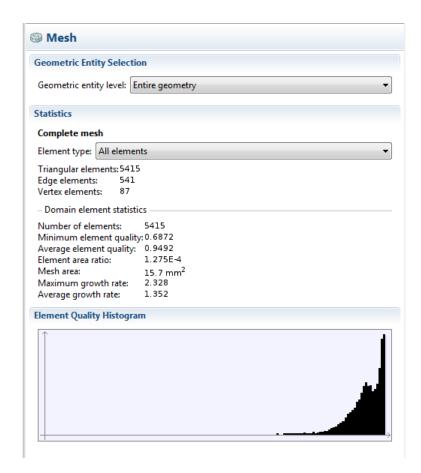


## Stress development design 2



## Some aspects of the mesh, design 1



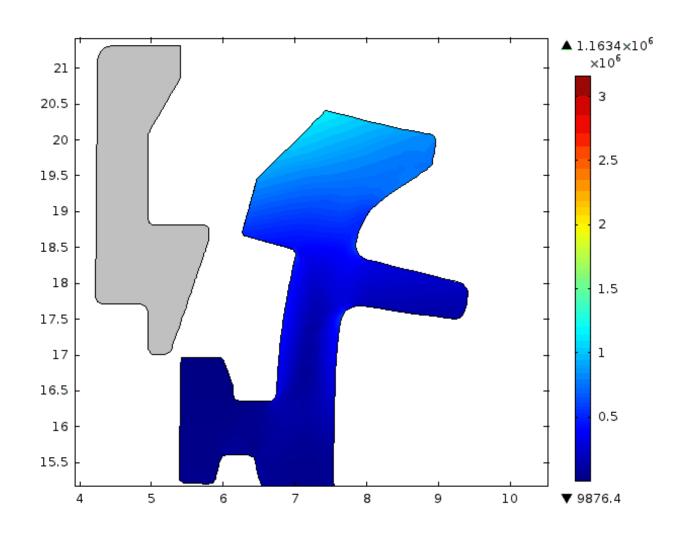


Final mesh properties.

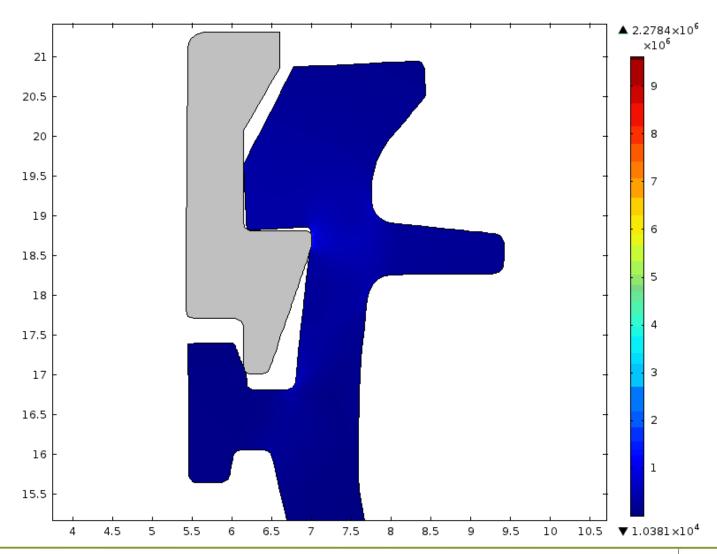
### Some remarks on simulation

- 2D axi-symmetric simulation.
- Mesh should be fine at contact points.
- Use small time steps, especially when stick slip occurs.
- Simulation time of 1 run approx. 1.5 hours
  - Can be optimised (Get the job done!)

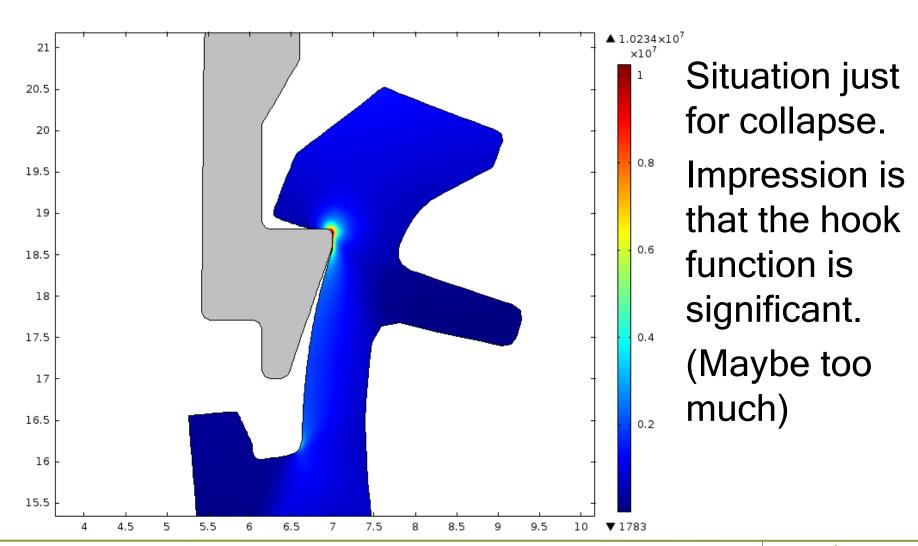
## **Assembly Design 1**



## Peel off, Design 1



## Design 1

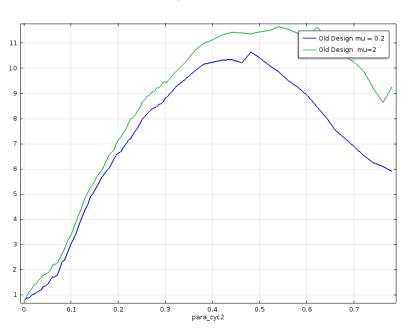


## Force on seal during peel off. Design 1 & 2

## Design 1

## 35 30 25 20 15 10 5 0 0.05 0.1 0.15 0.2 0.25 0.3 0.35 0.4 0.45 0.5

## Design 2



- Left design 1, Peak force can be seen.(μ=0.4!)
- Right design 2, Peak stress around 11- 14 N.

#### An overview of the two simulations

Solver time	Design 1	Design 2
Study 1: Assembly	52 sec	60 sec
Study 2: Pull off	70 min	87 min
	75 kDOF Axi symmetric	95 kDOF Axi symmetric
Mesh size		



### Some remarks

- With the friction coefficient of 2 (Laboratory measurements), the predicted maximum value of 11- 14N was measured in practice.
- A slight modification of the rounding of the metal ring was suggested for a smoother performance.
- Hyper-elastic module works well.
- Stick slip simulations are delicate, and take some time to study the behavior and tuning to get a stable solution.



## I would like to thank the support and measurements done by People Creating Value.

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Thanks to the People Creating Value

Thanks for your attention.