



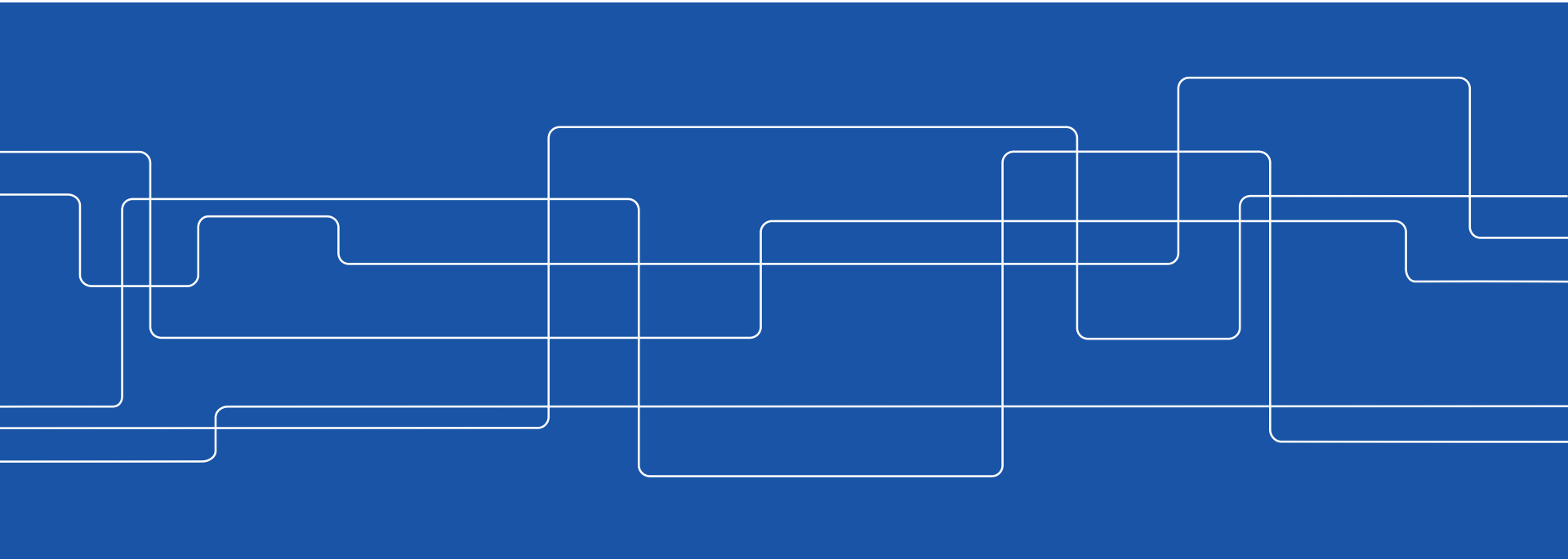
COMSOL CONFERENCE 2016 MUNICH



Acoustic Scattering through a Circular Orifice in Low Mach-Number Flow

Stefan Sack, Mats Åbom

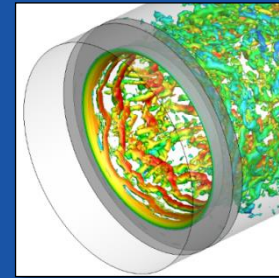
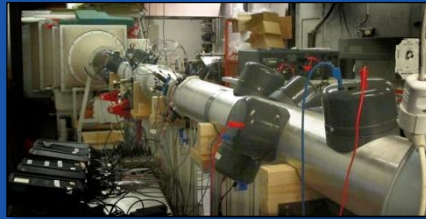
KTH, The Royal Institute of Technology, Stockholm, Sweden



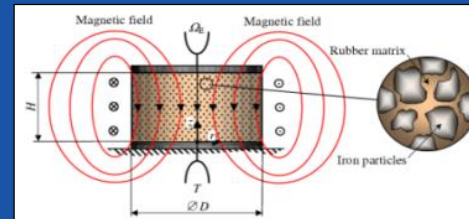
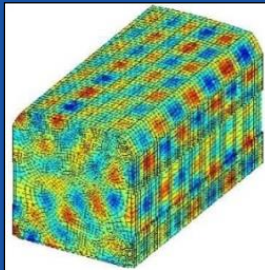
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Research Areas

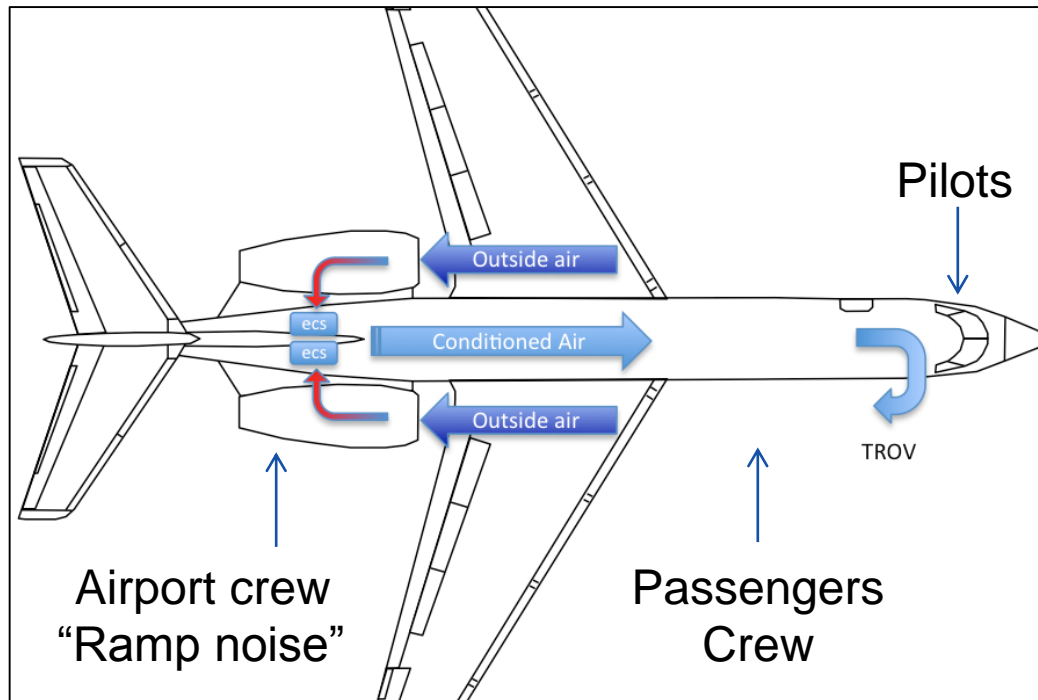


Aero-acoustics
Vibration isolators
Wave-based methods
Material acoustics



IdealVent Project

Integrated **Design** of Optimal **Ventilation** Systems for Low Cabin and Ramp Noise



Consortium

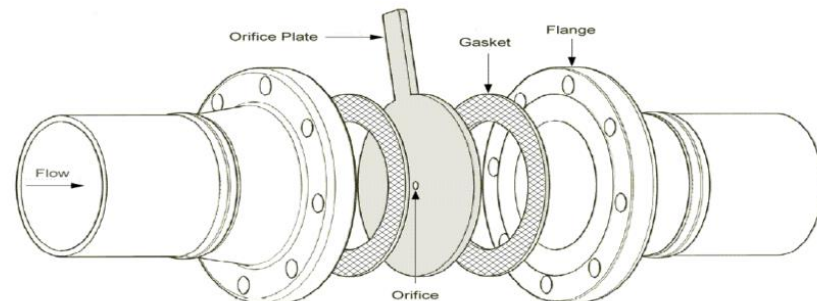
1. VKI - von Karman Institute for Fluid Dynamics, Belgium
2. DLR - Forschungszentrum der BRD für Luft- und Raumfahrt, Germany
3. KTH - Kungliga Tekniska Högskolan, Sweden
4. KUL - Katholieke Universiteit Leuven, Belgium
5. ECL - Ecole Centrale de Lyon, France
6. Siemens– (former LMS), Belgium
7. SNT - Odecon Sweden AB, Sweden
8. LTS - Liebherr Aerospace toulouse sas, France
9. NTS - New Technologies and services LLC, Russian Federation
10. EMB - Embraer S.A



Approach

A ventilation system contains (sophisticated) aero-acoustic sources (ducts, junctions, valves, nozzles, compressors, diaphragms)

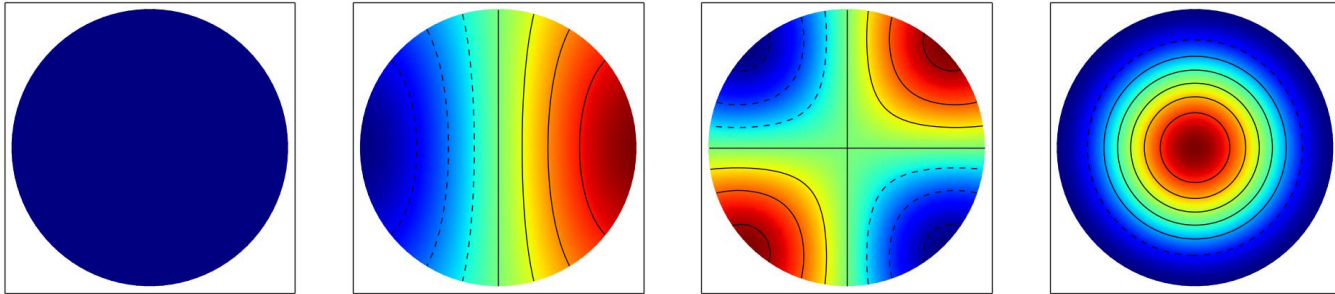
Idea: Investigate the sources separately and combine them to a network of so called multi-ports



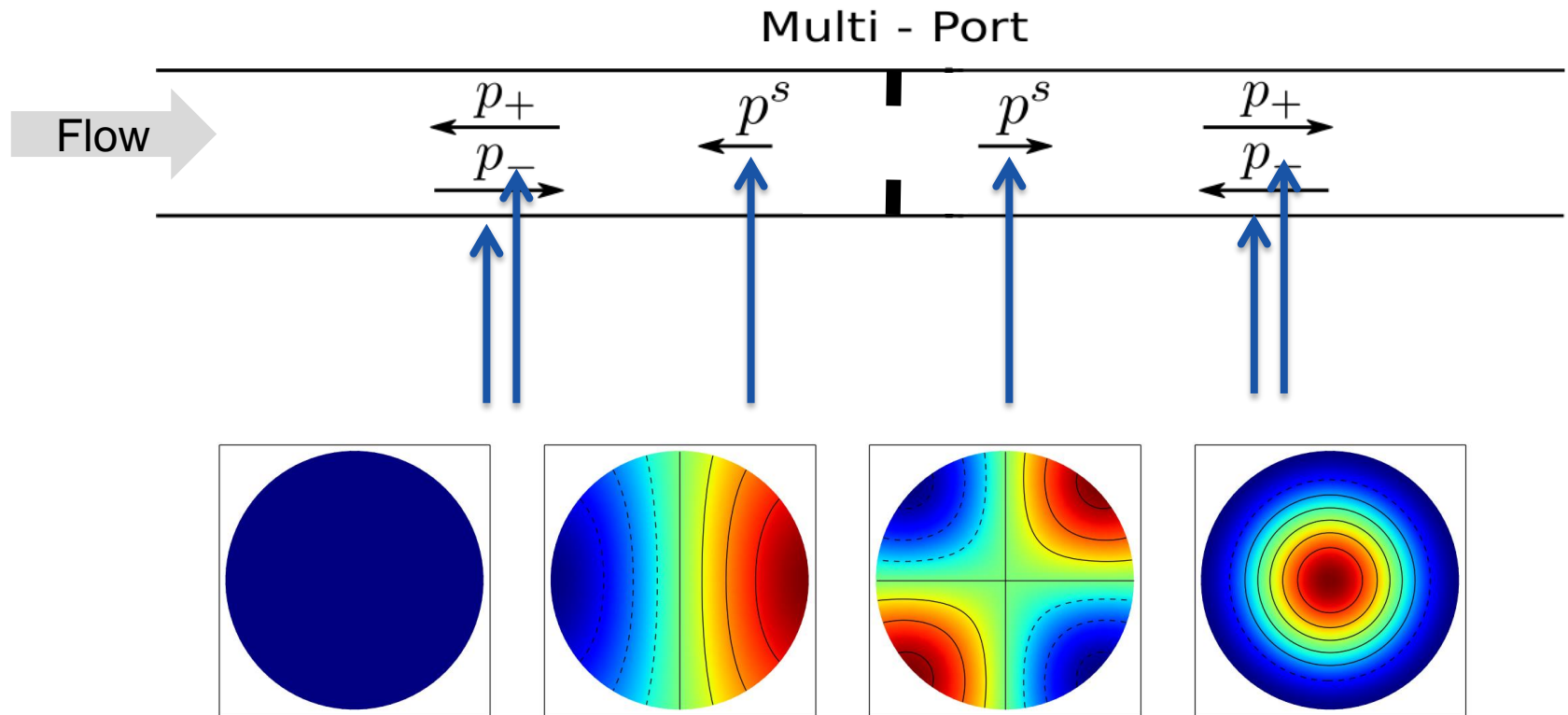
Multi-Port approach

The sound field inside the duct is a **superposition** of aeroacoustic **eigen-modes**

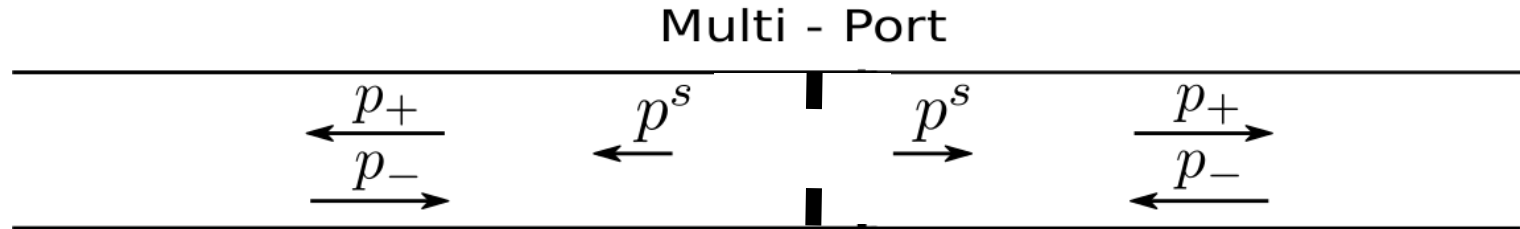
$$p(x, y, z) = \sum_{n=0}^N \hat{P}_n A_n(x, y, z)$$



Multi-Port approach



Multi-Port approach



We can find a linear system of equations

$$\mathbf{p}_+(f) = \mathbf{S}\mathbf{p}_-(f) + \mathbf{p}_+^s(f)$$

S Scattering Matrix

\mathbf{p}_+^s Source vector

See: **Lavrentjev, J. and Abom, M. (1996)**. *Characterization of Fluid Machines As Acoustic Multi-Port Sources*. Journal of Sound and Vibration, 197(1):1–16.

Multi - Port Networks

Orifice-plate in a circular duct

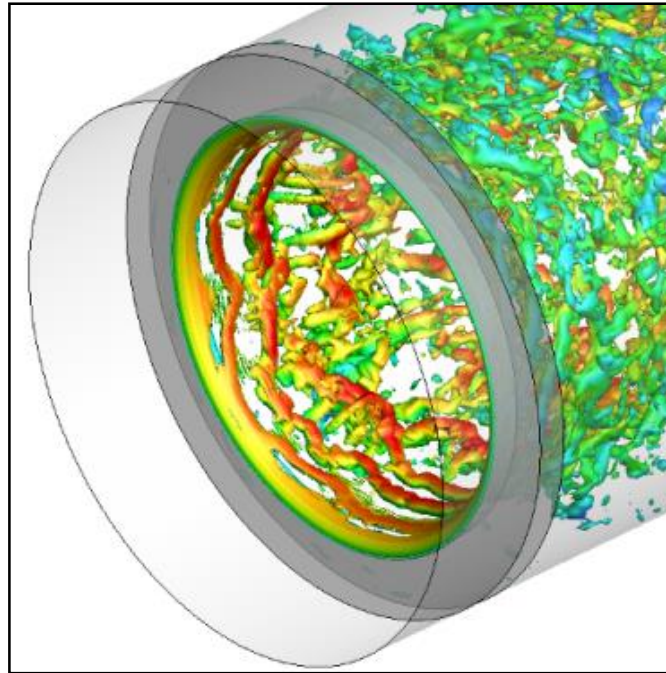
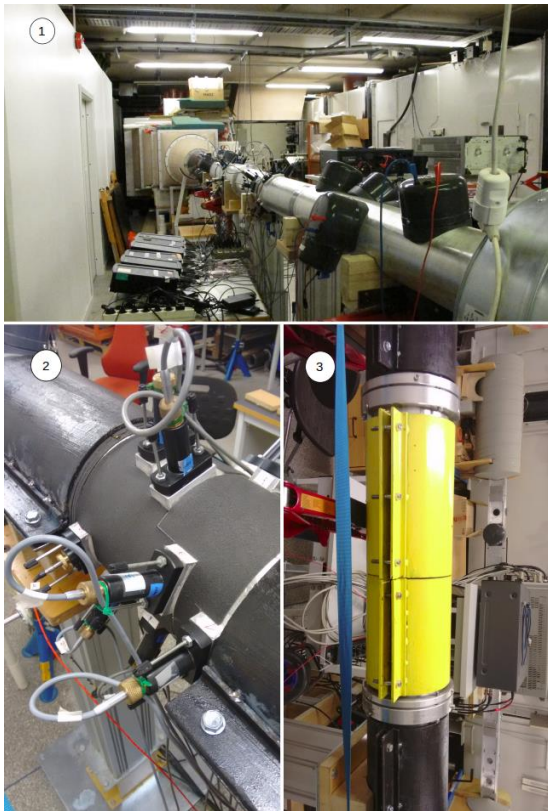


Figure from IDEES Computation by M. Shur et al., NTS

Measurements for model-validation at the *Marcus Wallenberg Laboratory for Sound and Vibration Research at KTH*



- **24 microphones** and **16 loudspeakers** in an optimised setup
- Aluminum pipe-sections with **constraint layer damping**
- Multi-channel excitation with algorithms for **simultaneous, uncorrelated excitation**
- **Modal decomposition** with **advanced** wave-numbers to account for damping
- Two stage measurements for **accurate scattering** and **source** characterisation

Sack, S., Åbom, M., & Efraimsson, G. (2016).

On Acoustic Multi-Port Characterisation Including Higher Order Modes.

Acta Acustica United with Acustica, 102, 834–850.

Computations

Linearized-Navier-Stokes equation

We use the **Linearized Navier-Stokes-Equation** in the **frequency domain** on a **3D grid**

$$i\omega\rho + \nabla \cdot (\rho\mathbf{u}_0 + \rho_0\mathbf{u}) = M$$

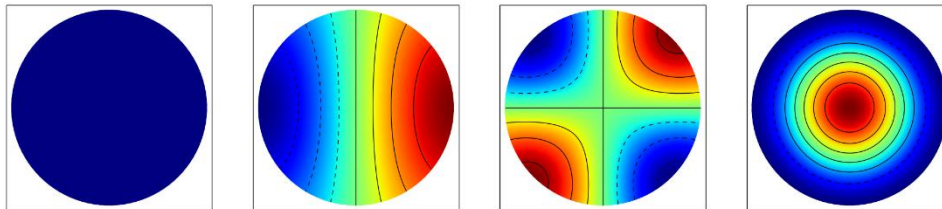
$$\rho_0(i\omega\mathbf{u} + (\mathbf{u} \cdot \nabla)\mathbf{u}_0 + (\mathbf{u}_0 \cdot \nabla)\mathbf{u}) - \nabla \cdot \boldsymbol{\sigma} = \mathbf{F}$$

$$\rho_0 C_p (i\omega T + (\mathbf{u} \cdot \nabla)T_0 + (\mathbf{u}_0 \cdot \nabla)T) + \rho C_p (\mathbf{u}_0 \cdot \nabla)T_0$$

$$-\alpha_0 T_0 (i\omega p + (\mathbf{u} \cdot \nabla)p_0 + (\mathbf{u}_0 \cdot \nabla)p) - \alpha_0 T (\mathbf{u}_0 \cdot \nabla)p_0 - \nabla \cdot (k\nabla T) = \Phi + Q$$

$$\boldsymbol{\sigma} = -p\mathbf{I} + \mu(\nabla\mathbf{u} + (\nabla\mathbf{u})^T) + \left(\mu_B - \frac{2}{3}\mu\right)(\nabla \cdot \mathbf{u})\mathbf{I}$$

$$\rho = \rho_0(\beta_T p - \alpha_0 T)$$

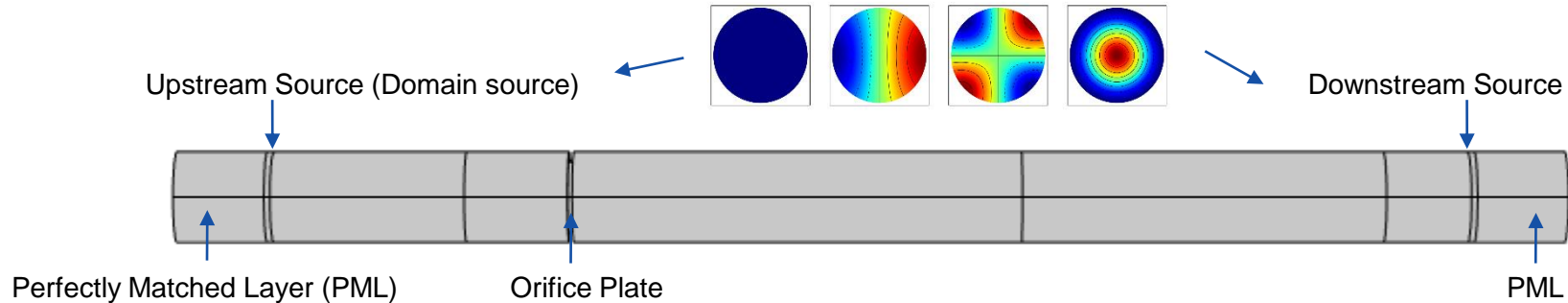


- Orifice 3D (comp3)
 - Definitions
 - Boundary System 3 (sys3)
 - Perfectly Matched Layer 3 (pml3)
 - View 27
 - Geometry 3
 - Materials
 - Air (mat3)
 - Linearized Navier-Stokes, Frequency Domain (lnsf)**
 - Linearized Navier-Stokes Model 1
 - Wall 1
 - Initial Values 1
 - Wall 2
 - Domain Sources 1
 - Domain Sources 2
 - Linearized Navier-Stokes Model 2
 - Equation View

For Linearized-Navier-Stokes solver for acoustic plane wave modes see: **Kierkegaard et al. (2012)**. Simulations of whistling and the whistling potentiality of an in-duct orifice with linear aeroacoustics. Journal of Sound and Vibration.

Computations

Linearized-Navier-Stokes equation



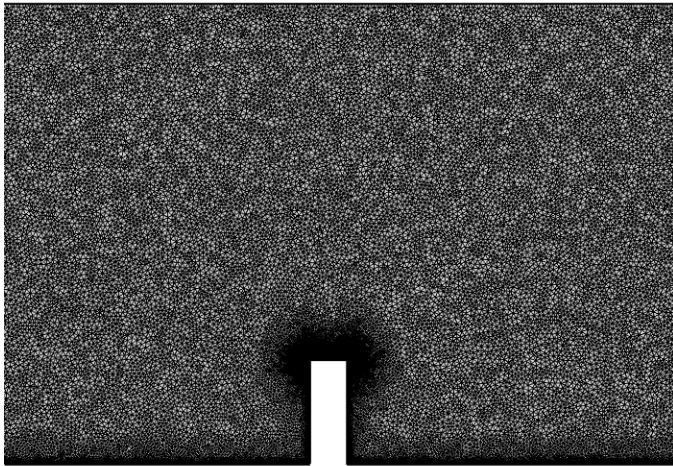
Computational Steps for scattering:

1. Background mean flow (pref. SST RANS), once
2. Acoustics, one computation for each mode and upstream and downstream side per frequency

Computations

Linearized-Navier-Stokes equation

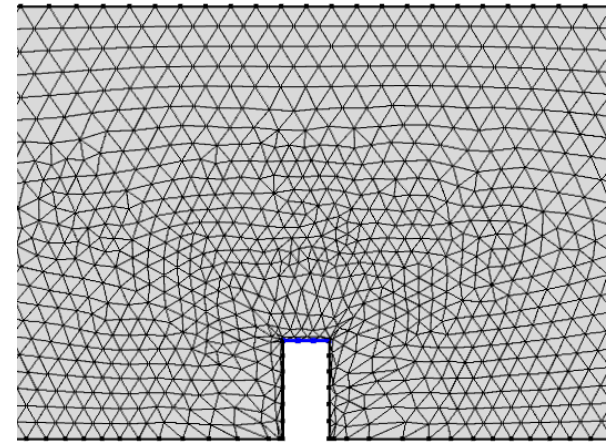
Mesh: Flow computation



Interpolation



Mesh: Acoustic computation



Problem: The internal interpolation seems to be insufficient, as the gradients are not interpolated correctly on the language points inside the elements

Solution: “Dummy-study” for interpolation using weak formulation to minimize difference between flow and interpolated solution.



Computations Mean Flow



Problem with flow computation: No convergence for SST RANS, even in 2D

Tried different “tricks”:

K-Epsilon / K-omega start solution

Inconsistent stabilisation

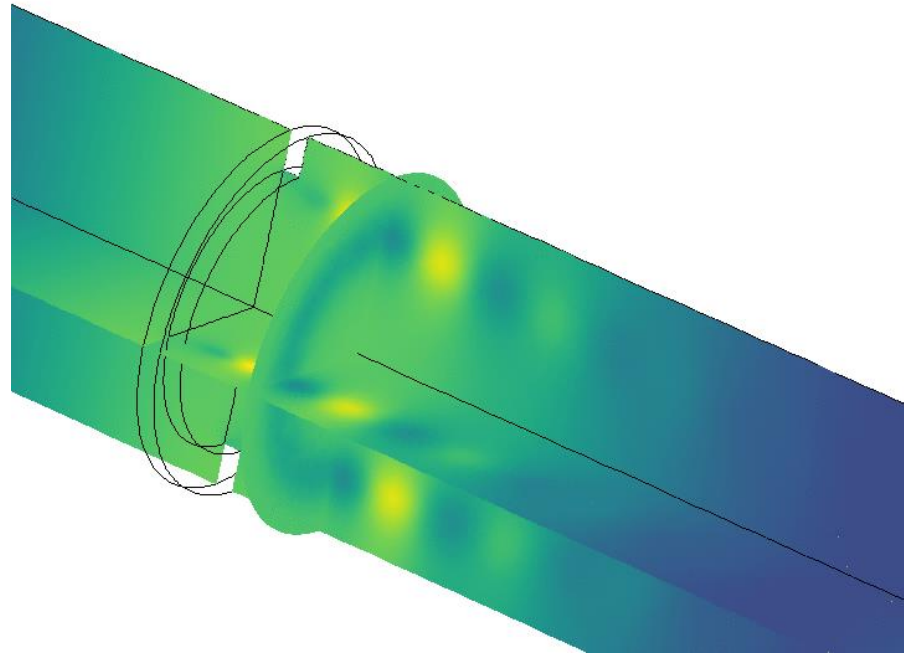
Pseudo time stepping

Smooth corners

No solution. Approach: Compute flow in different software and import it into Comsol.
(this is not optimal, any ideas are more than appreciated)

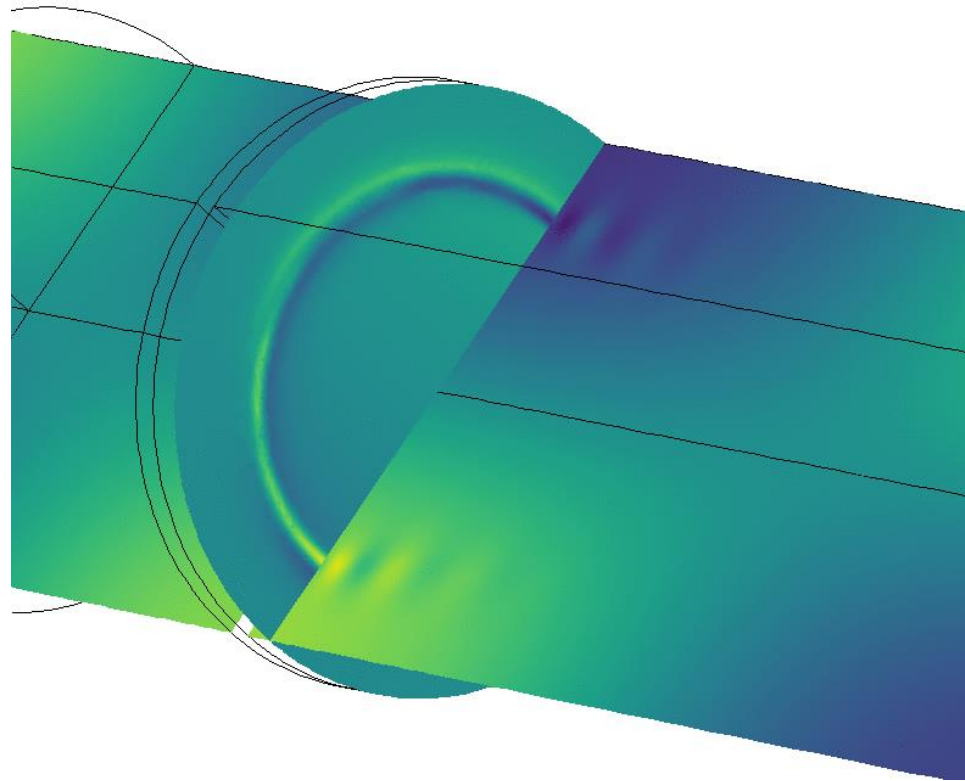
In general, acoustic computations worked very well!

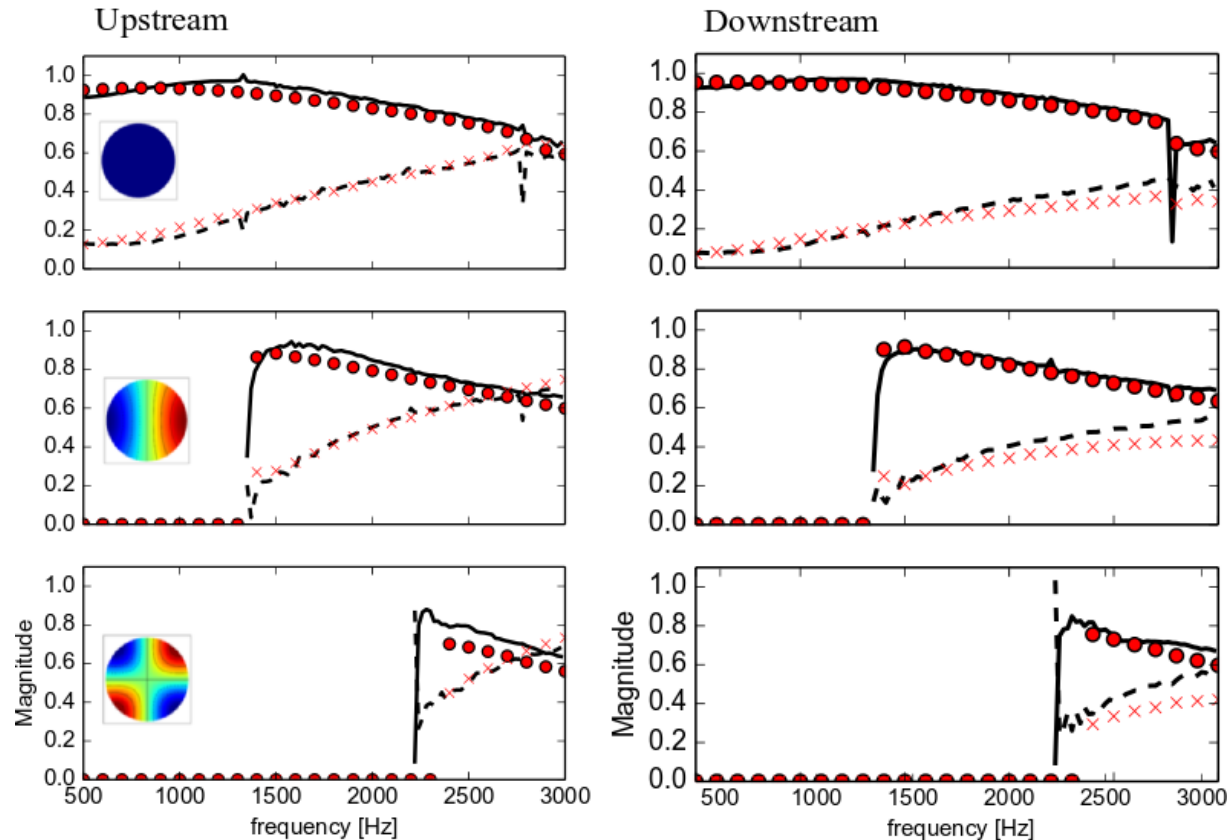
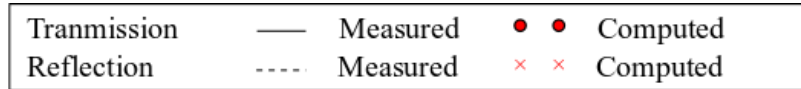
Density fluctuation
(0,0)-Mode



In general, acoustic computations worked very well!

Density fluctuation
(1,0)-Mode





Summery

- The linearized Navier-Stokes Interface can be used to compute the scattering of sound including effects of acoustic-flow interaction
- For circular orifice plates, the crucial point seems to be computing the mean flow, esp. the thin shear layers in the jet downstream the orifice
- If the mean-flow is computed and interpolated, very accurate results can be achieved with rather low computational costs



Full Multi-Port Characterization of a Circular Orifice Plate

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